

Accuracy and Composability in NODAS

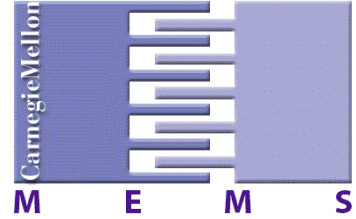
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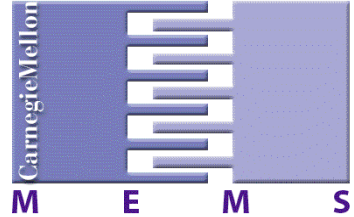
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<http://www.ece.cmu.edu/~mems>**

Outline



- Introduction
 - MEMS Simulation with NODAS
- Gap Model
 - Model Formulation
 - Contact Mechanism
 - Lateral Motion
- Accuracy and Composability
- Future Work

Introduction

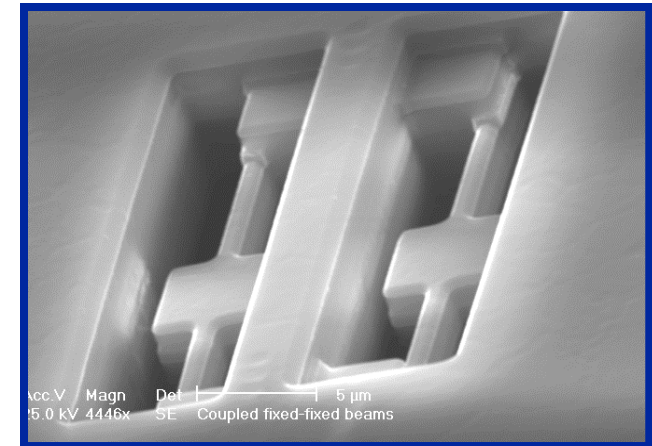


- **NODAS (NOdal Design of Actuators and Sensors)**
 - **Library of composable MEMS atomic elements**
 - **Beams**
 - **Plates**
 - **Electrostatic Gaps**
 - **Simulation in Cadence Spectre**
 - **Electrical and mechanical co-simulation**
- **Examples of Related Work:**
 - **SUGAR, UC Berkeley**
 - **ARCHITECT, Coventor**
 - **Work by G. Lorenz and R. Nuel, Bosch**

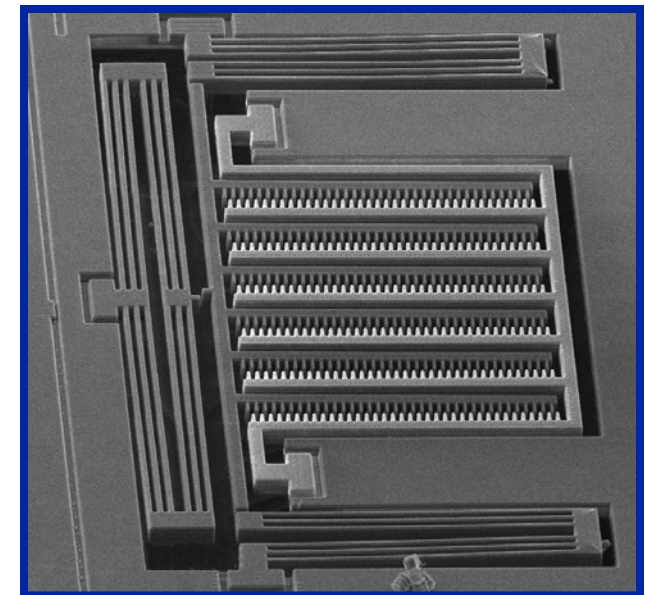
Composable MEMS Models

- Examples:
 - RF Switches
 - Resonant mixers/filters
 - Variable Capacitors

- Mechanical Models
 - Beams
 - Plates
- Electromechanical Model
 - Electrostatic Gaps



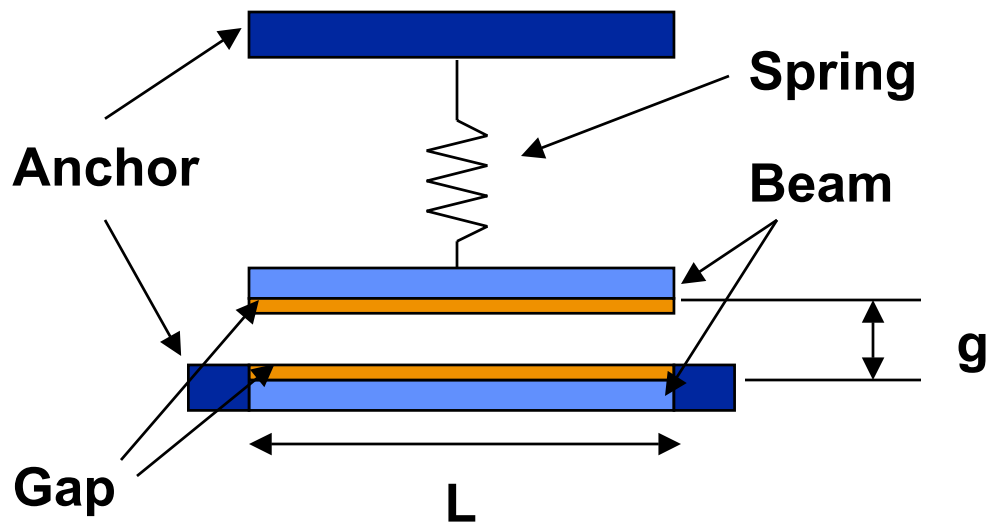
MEMS Resonant Mixer



MEMS Varactor

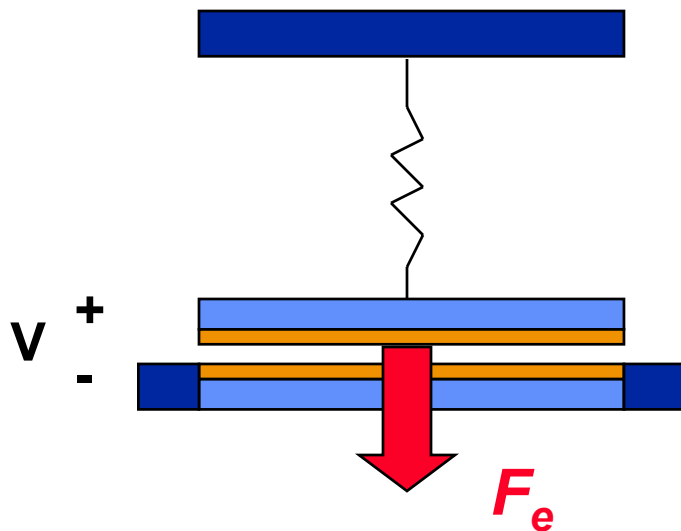
Canonical Problem

- Voltage across air gap
 - Generates electrostatic force
 - Displaces moveable electrode towards fixed electrode



Canonical Problem

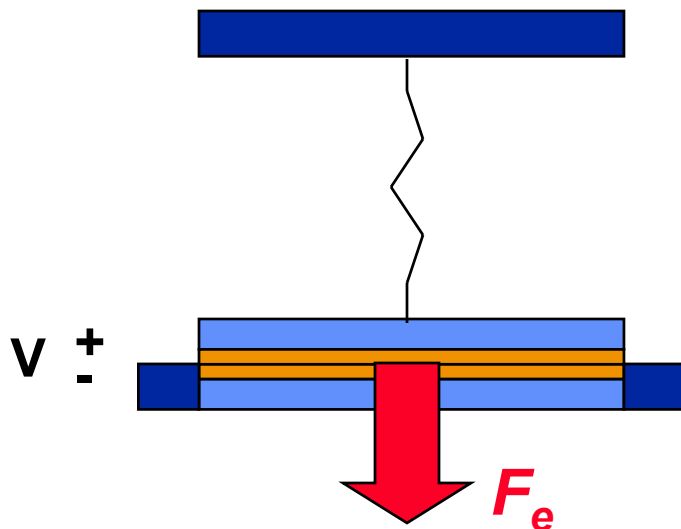
- Voltage across air gap
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$$F_e = \frac{1}{2} \frac{\epsilon_0 \cdot V^2 \cdot L \cdot T}{g^2}$$

Canonical Problem

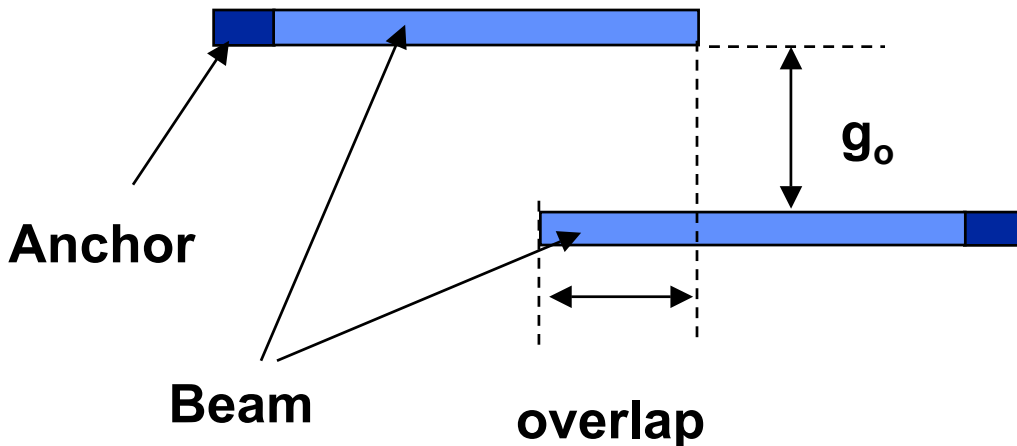
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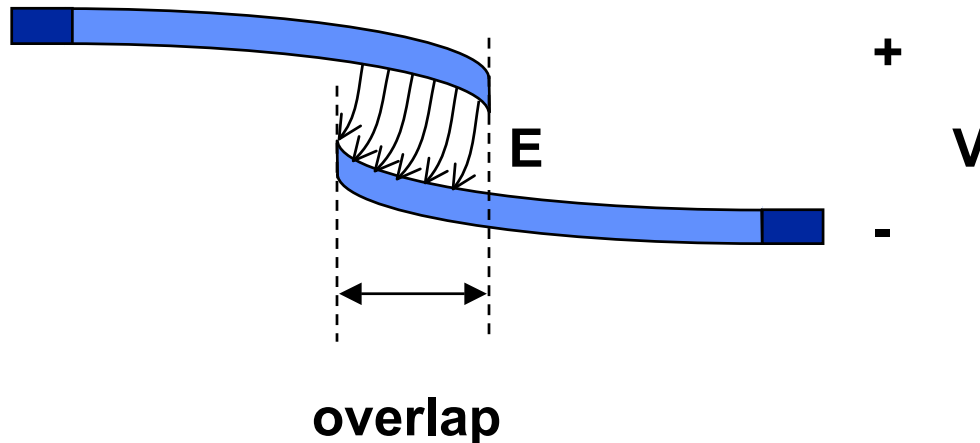
Model Formulation with Beam Electrodes

- Two cantilever beams separated by a gap g_0



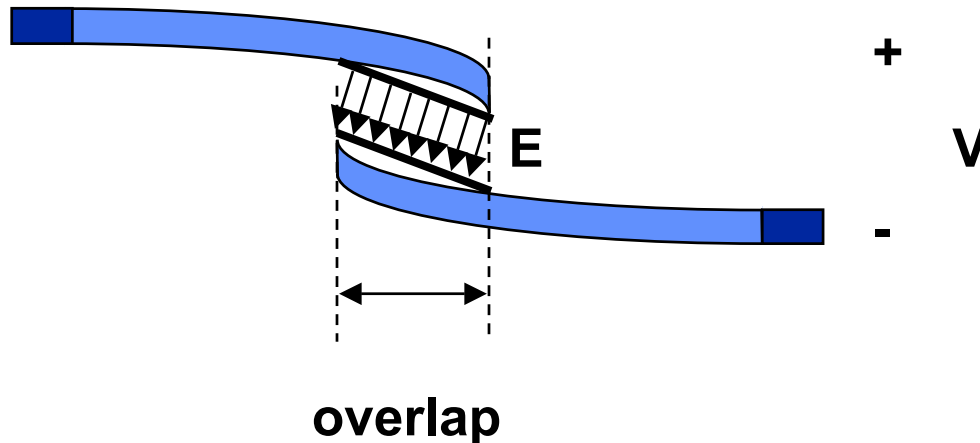
Model Formulation with Beam Electrodes

- Two cantilever beams separated by a gap g_0
- Apply voltage V :
 - Generates electric field E
 - Bends beams toward each other
- Difficult problem: no closed form solution



Model Formulation with Beam Electrodes

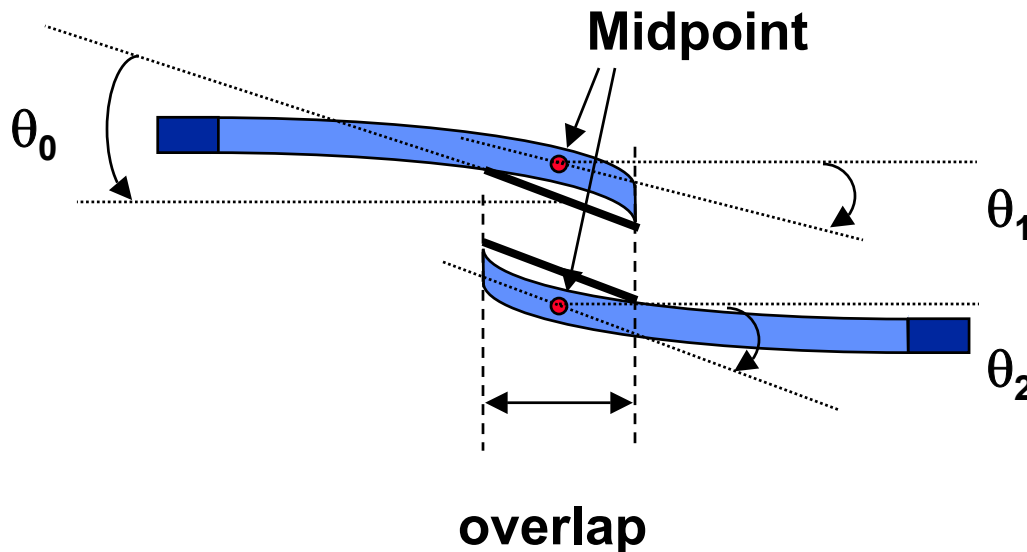
- Two cantilever beams separated by a gap g_0
- Apply voltage V :
 - Generates electric field E
 - Bends beams toward each other
- Difficult problem: no closed form solution
- **Approximate with rotated rigid parallel plates**



Rotated Parallel Plate

- Calculate rotation angle
 - Average angle at the midpoint of overlap region

$$\text{theta0} = (\text{ang1mid} + \text{ang2mid}) / 2.0;$$



$$\theta_0 = \frac{\theta_1 + \theta_2}{2}$$

Rotated Parallel Plate

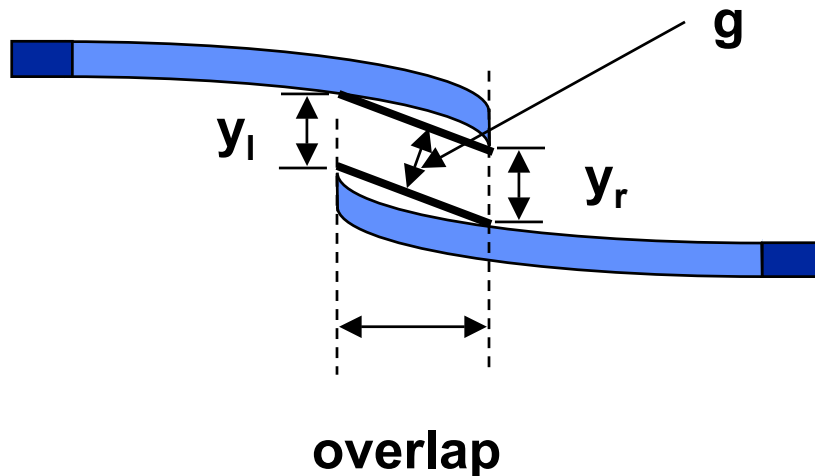
- Calculate rotation angle
 - Average angle at the midpoint of overlap region

■ Calculate gap

$$g = \cos(\theta_0) * \min(y_l, y_r)$$

$$dy_l = \min(\overbrace{(y_{1r} - y_{2r} + gap)}^{y_r}, \overbrace{(y_{1l} - y_{2l} + gap)}^{y_l});$$

$$dy = dy_l * \cos(\theta_0);$$

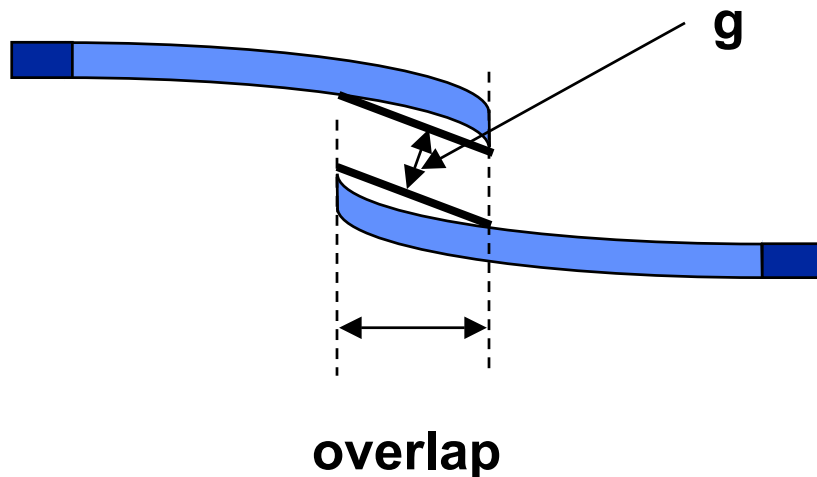


Rotated Parallel Plate

- Calculate rotation angle
 - Average angle at the midpoint of overlap region
- Calculate gap

$$g = \cos(\theta_0) * \min(y_l, y_r)$$
- Calculate electrostatic force/length in this rotated frame

$$F_{elec} = -0.5 * \epsilon_0 * thickness * v_squared / pow(dy, 2) ;$$

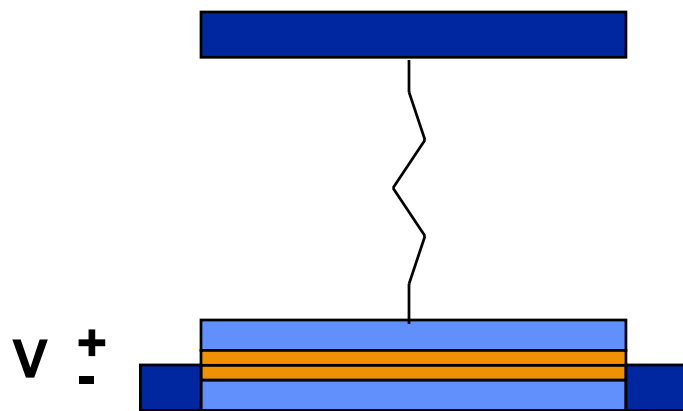


$$F_e = \frac{1}{2} \frac{\epsilon_0 \cdot V^2 \cdot overlap \cdot T}{g^2}$$

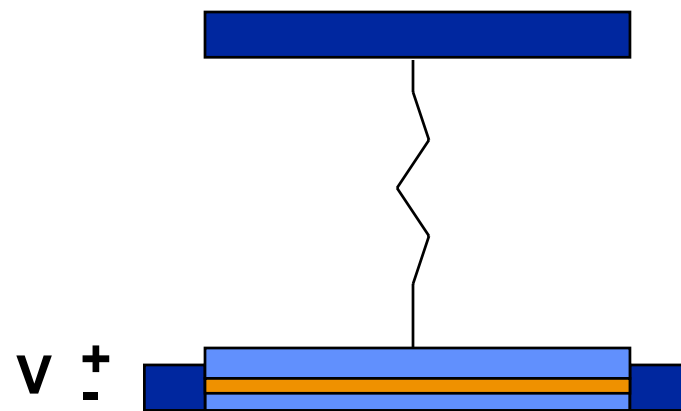
Contact Model

■ Contact Model

- Displacement beyond fixed electrode is non-physical ($g < 0$)
- Contact force model prevents this



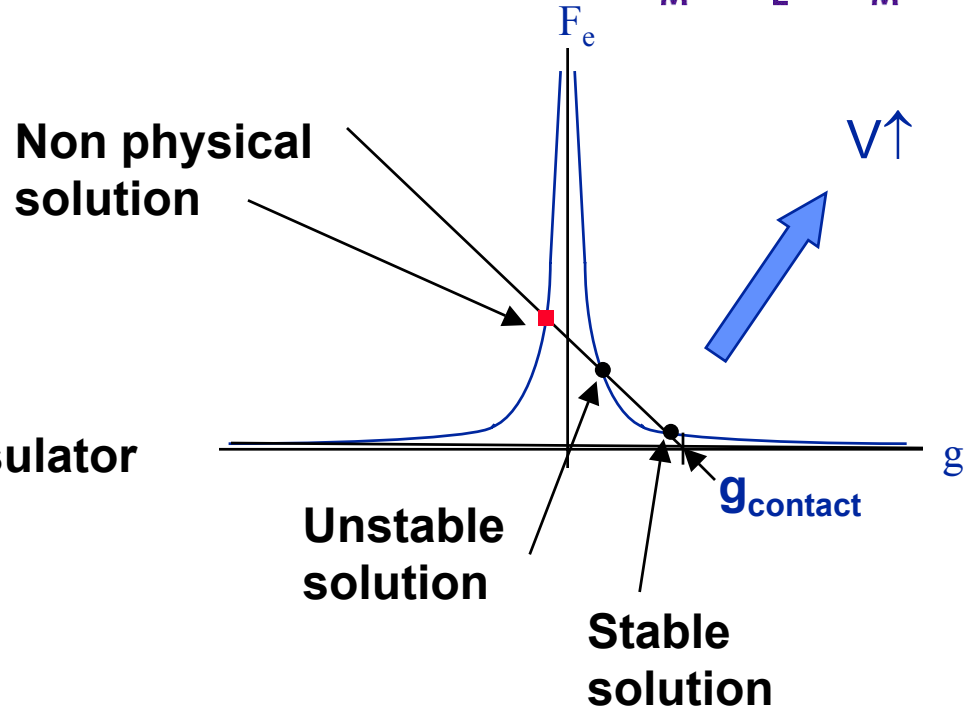
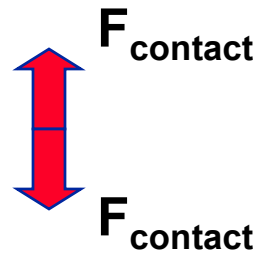
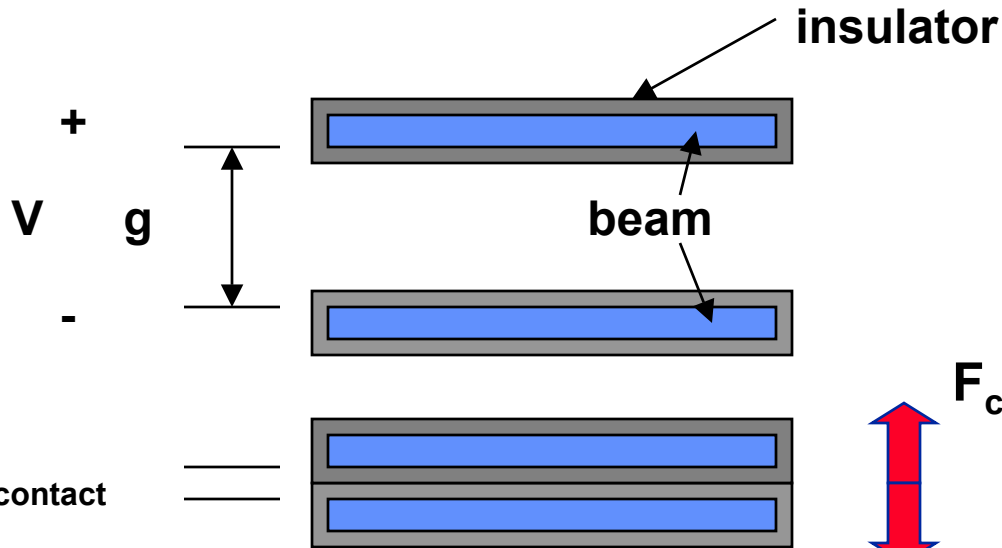
Beams in contact



Non-physical solution

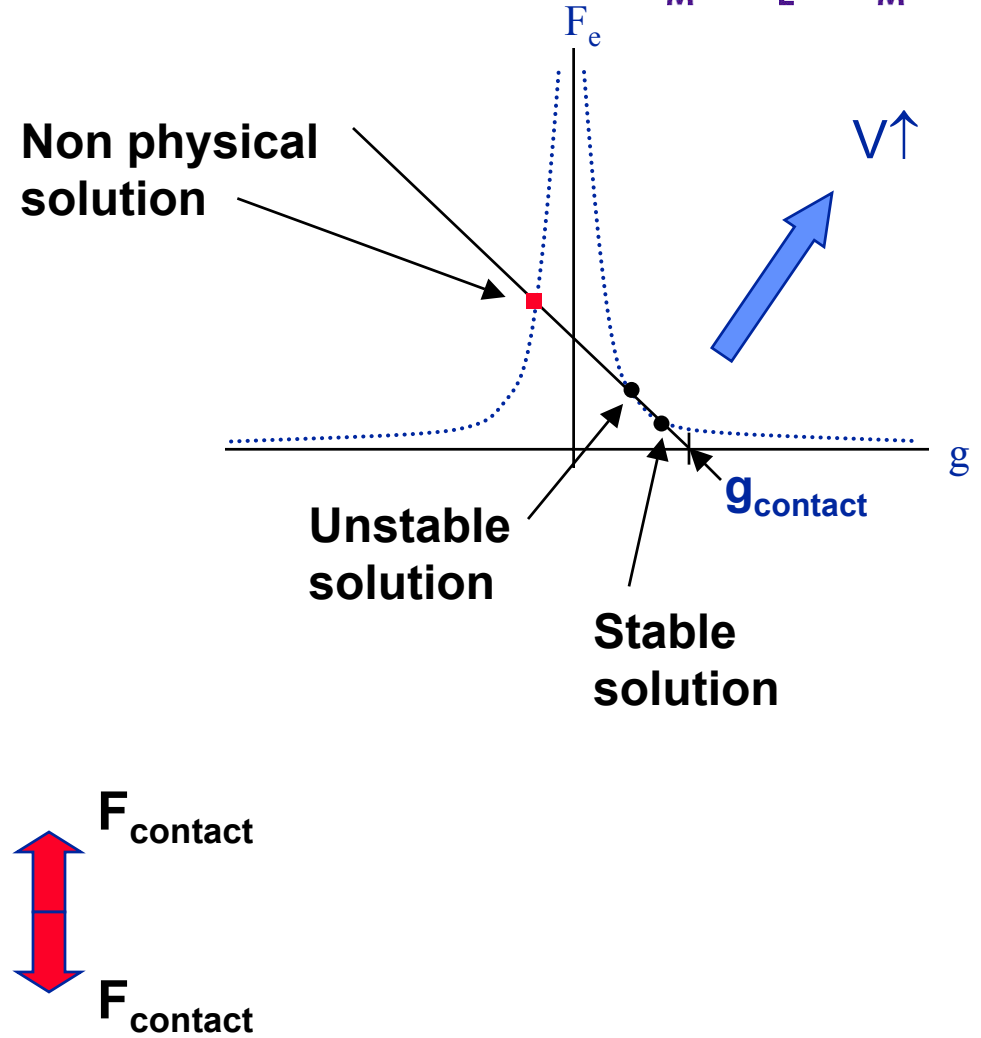
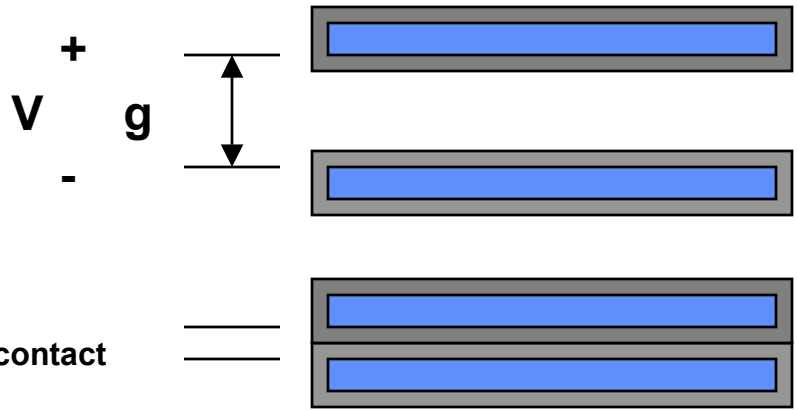
Gap Behavior in Contact

$$F_e = \frac{1}{2} \frac{\epsilon_0 \cdot V^2 \cdot L \cdot T}{g^2}$$

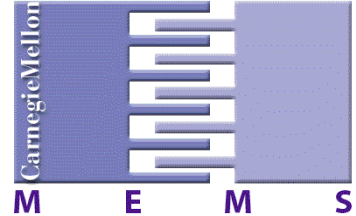


Gap Behavior in Contact

$$F_e = \frac{1}{2} \frac{\epsilon_0 \cdot V^2 \cdot L \cdot T}{g^2}$$

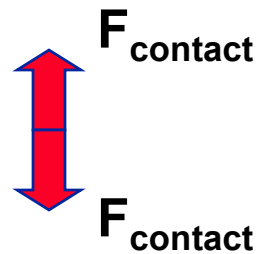
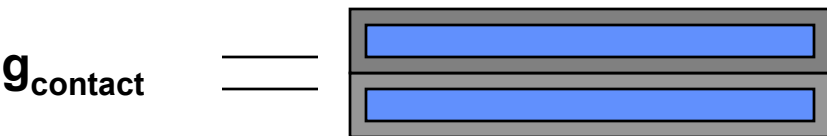
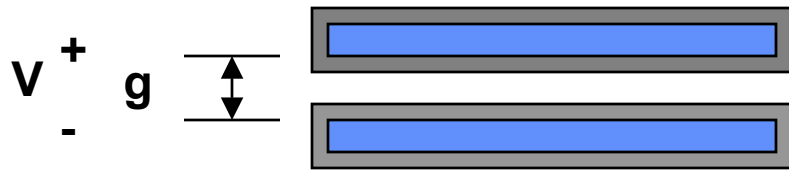
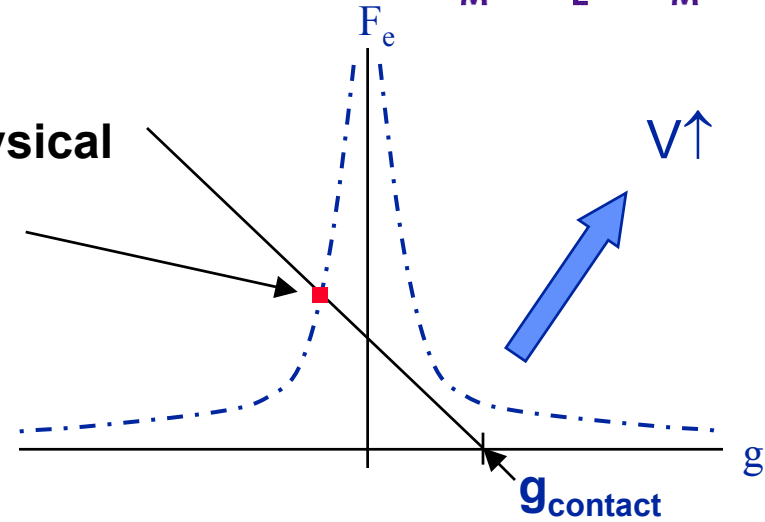


Gap Behavior in Contact



$$F_e = \frac{1}{2} \frac{\epsilon_0 \cdot V^2 \cdot L \cdot T}{g^2}$$

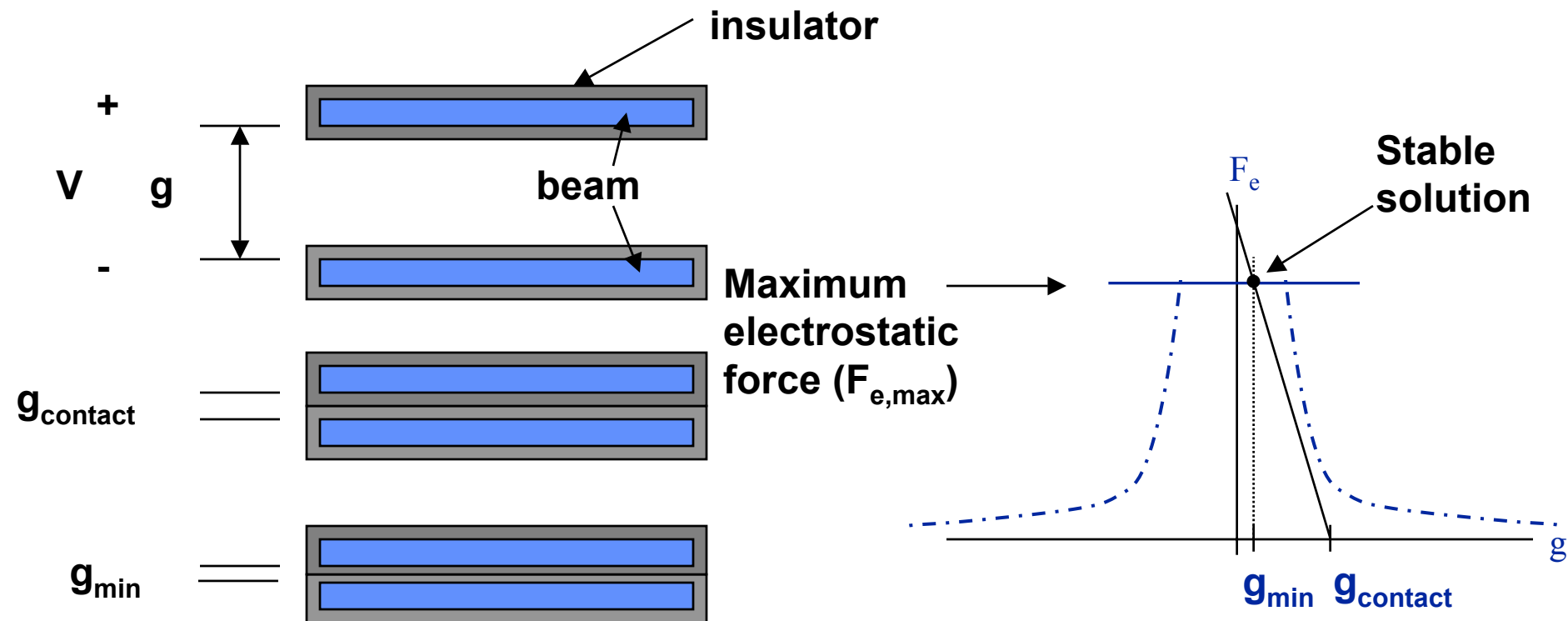
Non physical solution



Gap Behavior in Contact

$$F_e = \frac{1}{2} \frac{\epsilon_0 \cdot V^2 \cdot L \cdot T}{g^2}$$

$$F_{e,max} = \frac{1}{2} \epsilon_0 \cdot E_{BD}^2 \cdot L \cdot T$$

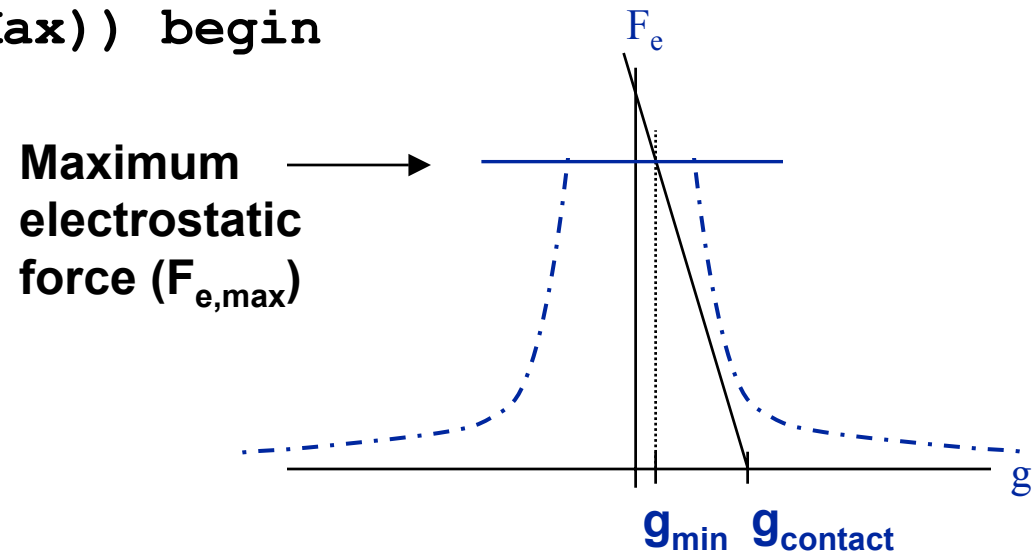


Contact Model Verilog-A

```

FeMax = - 0.5*`eps0*thickness*pow(elec_breakdown,2);
slope = FeMax/(gmin - gcontact);
if (dy<gcontact) begin
    Fy_contact = slope*(dy-gcontact);
end
else begin
    Fy_contact = 0;
end
if (abs(Felec) > abs(FeMax)) begin
    Fl = FeMax;
end
else begin
    Fl = Felec;
end

```



Snap In

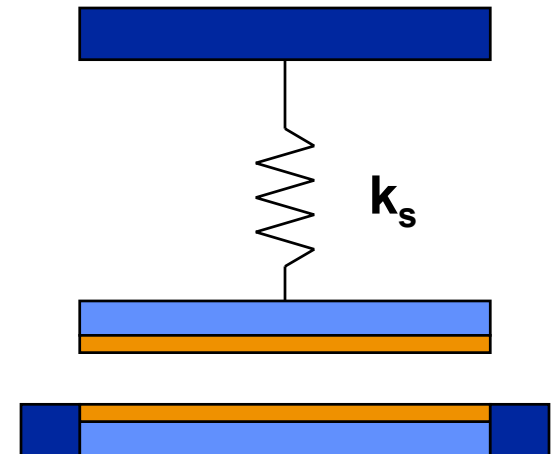
- System unstable when $k_e > k_s$
- Gap snaps in at 2/3 of the original gap
- Pull in voltage, $V_{PI} = 67.47$ V for this case

$$F_e = \frac{1}{2} \frac{\epsilon_0 \cdot V^2 \cdot L \cdot T}{(g_o - y)^2} \quad k_e = \frac{dF_e}{dy} = \frac{\epsilon_0 \cdot V^2 \cdot L \cdot T}{g^3}$$

$$F_s = k_s \cdot y$$

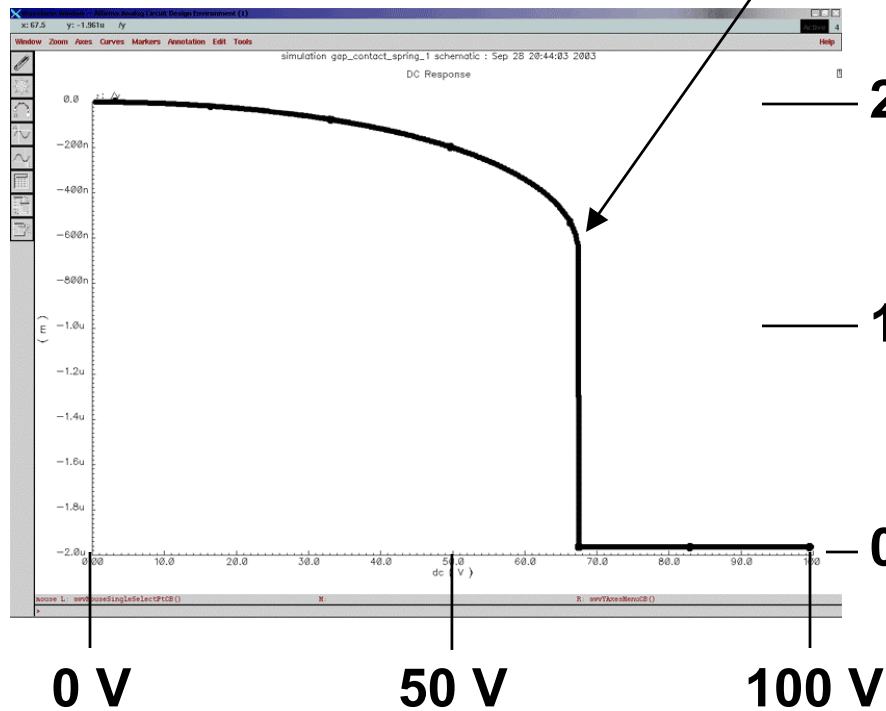
$$g_{PI} = \frac{2}{3} g_o$$

$$V_{PI} = \sqrt{\frac{k_s \cdot g_{PI}^3}{\epsilon_0 \cdot L \cdot T}}$$

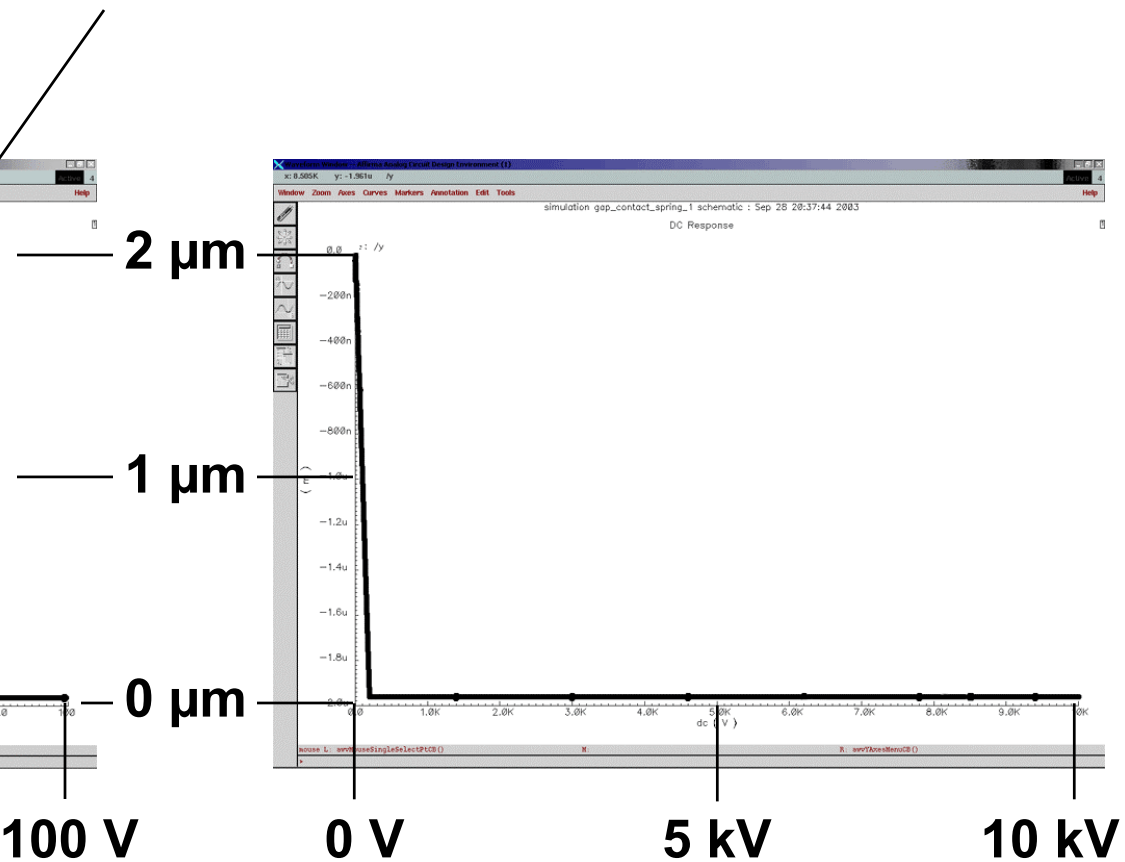


Snap In/Contact Simulation

- Physical solution found
- Snap in occurs at $V = 67.5 \text{ V}$



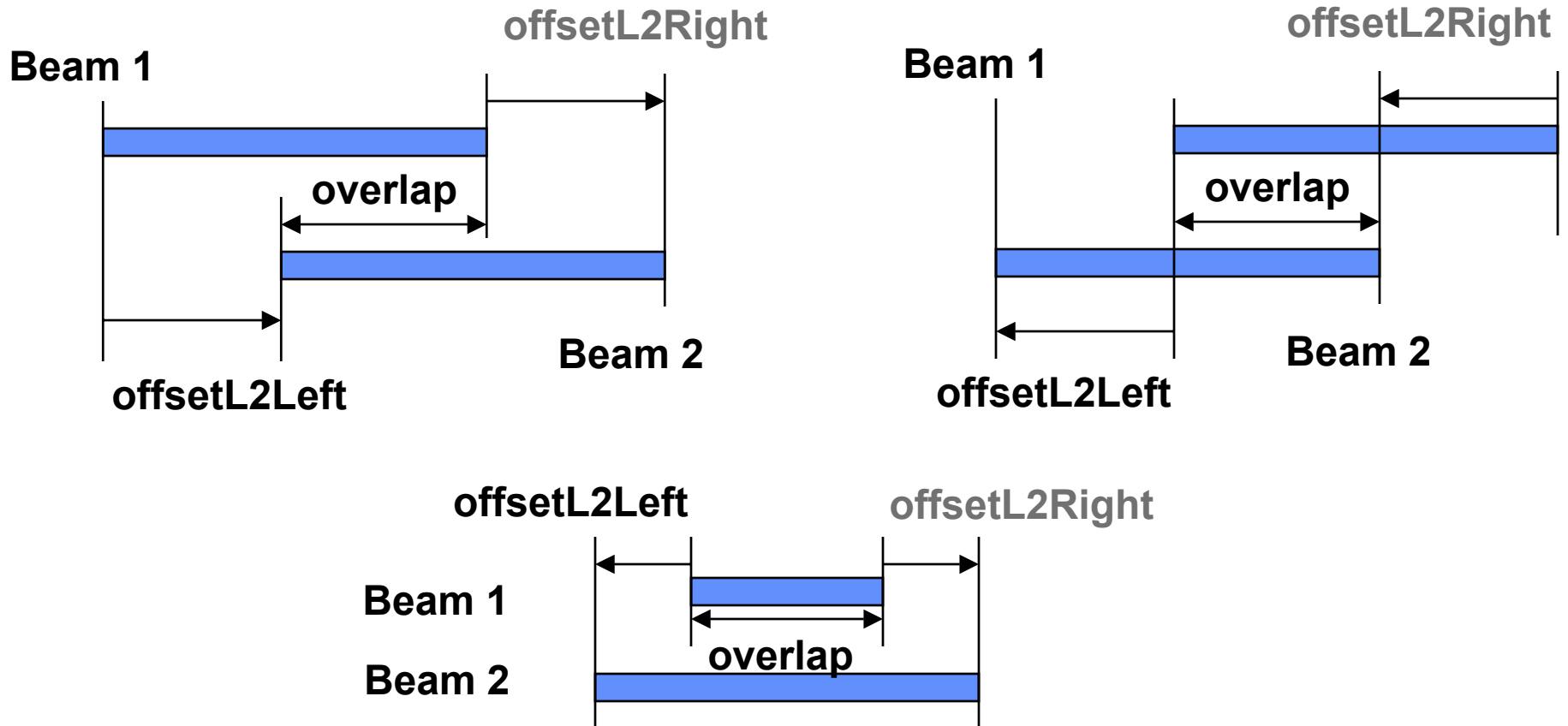
0 V - 100 V DC Sweep



0 V - 10000 V DC Sweep

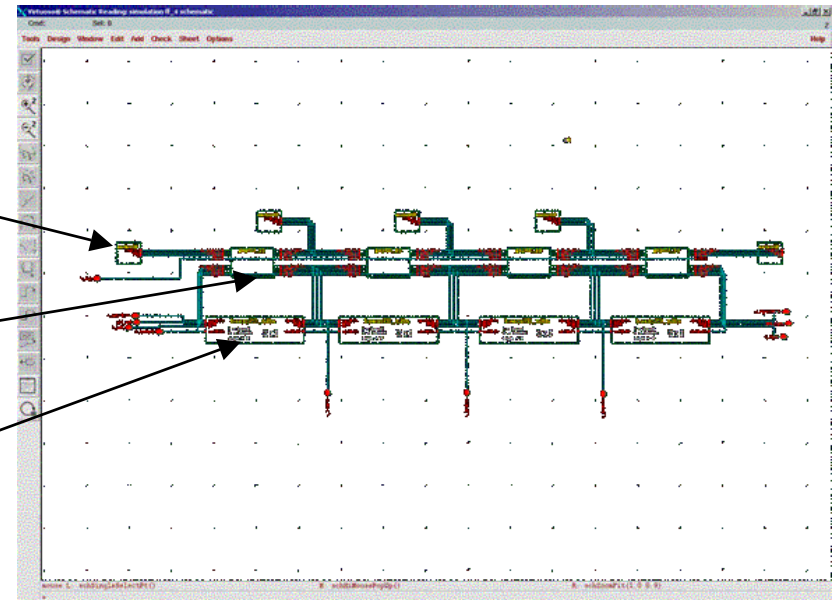
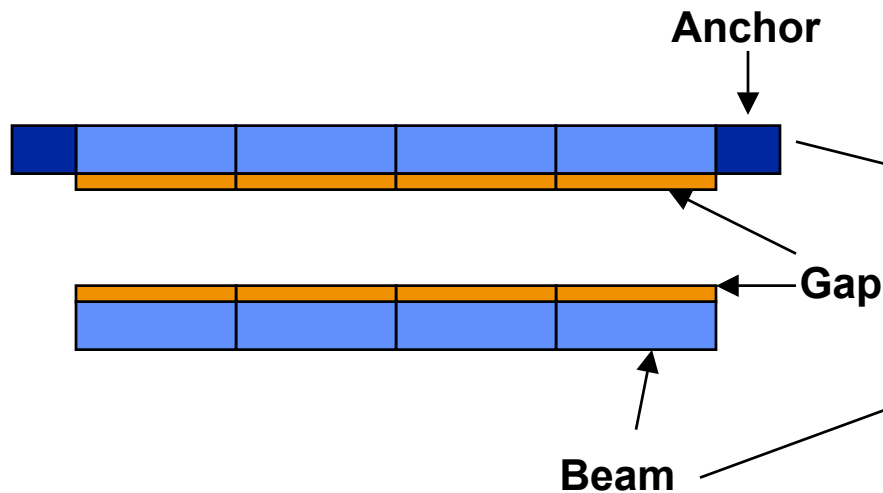
Lateral Motion

- Electrostatic force in x direction
 - Displaces electrodes laterally



NODAS Composability

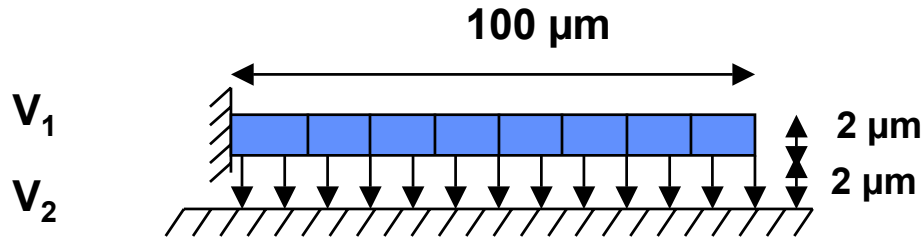
- NODAS model
 - Decompose beam into 2, 4, 8, 16, and 32 segments
- Verification: by comparison with FEMLAB



4 beam/gap segments

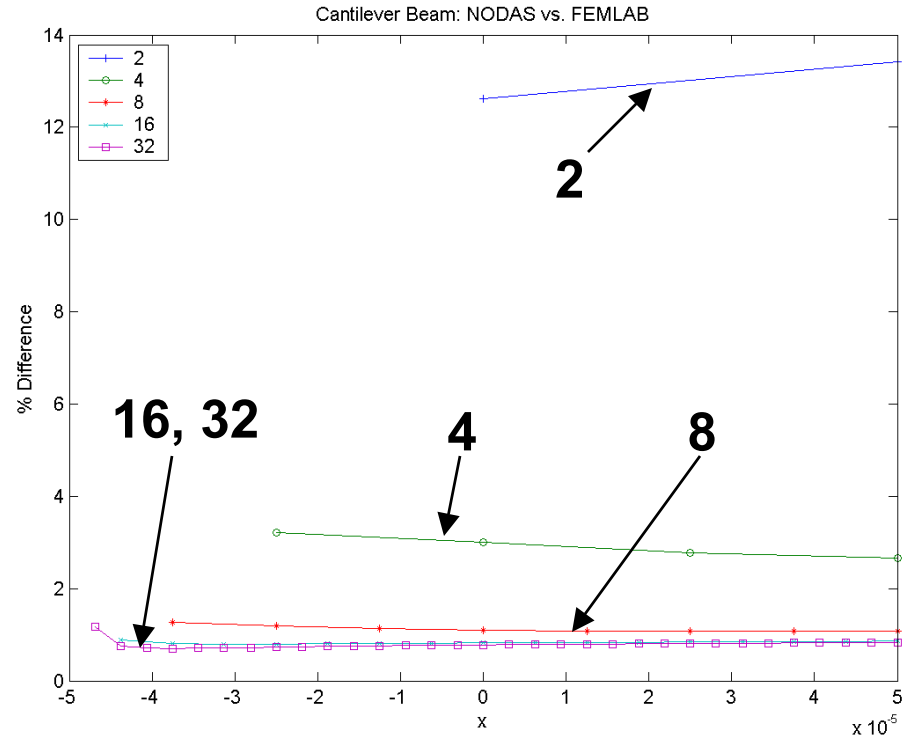
4 beam/gap segments schematic

Electrostatically Actuated Cantilever

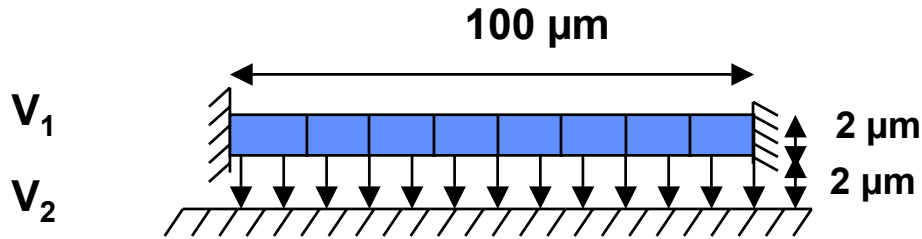


$E = 170 \text{ GPa}$ $t = 2 \mu\text{m}$
 Density = 2330 kg/m^3

- Comparison between NODAS and FEMLAB
 - Large error for 2 segments
 - $< 1\%$ difference for ≥ 16 segments

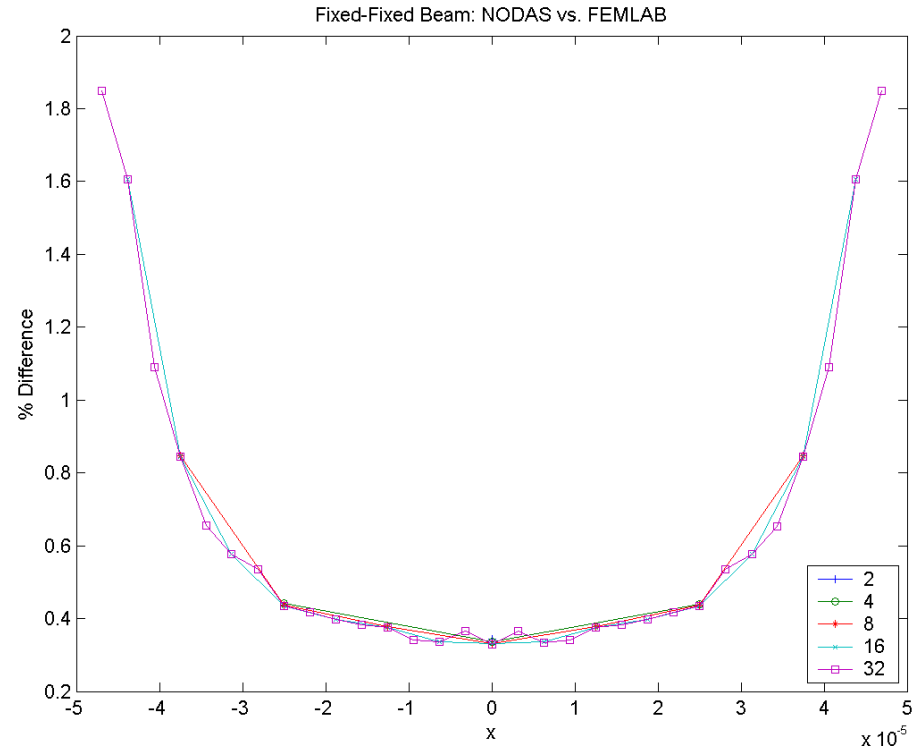


Electrostatically Actuated Fixed-Fixed Beam



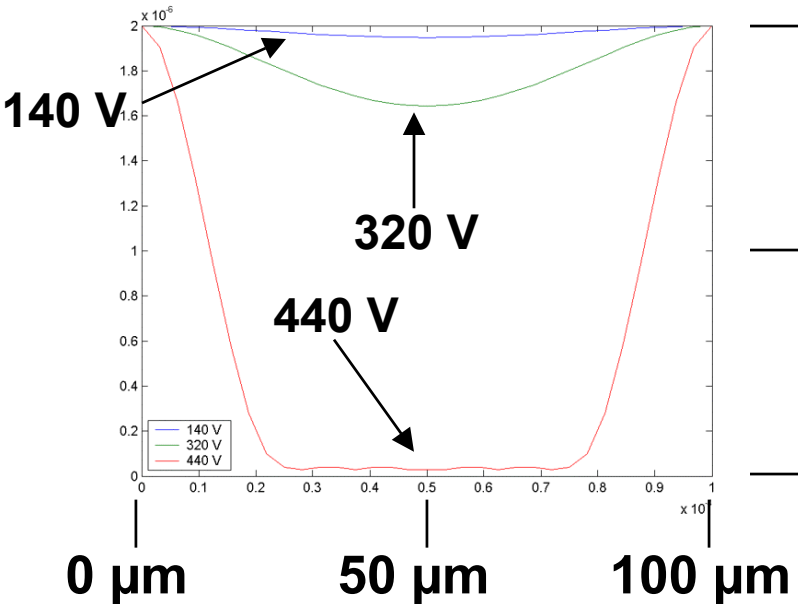
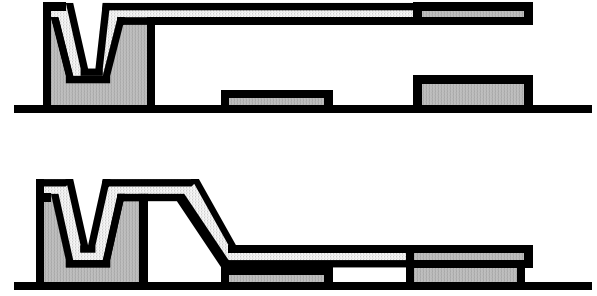
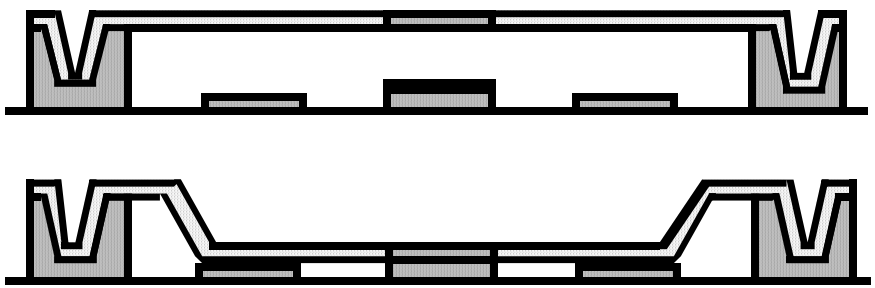
$E = 170 \text{ GPa}$ $t = 2 \mu\text{m}$
 Density = 2330 kg/m^3

- Comparison between NODAS and FEMLAB
 - Error greatest at anchored ends (<1% if NODAS shear model used)
 - < 0.5% error at beam center



RF Switch Snap-In

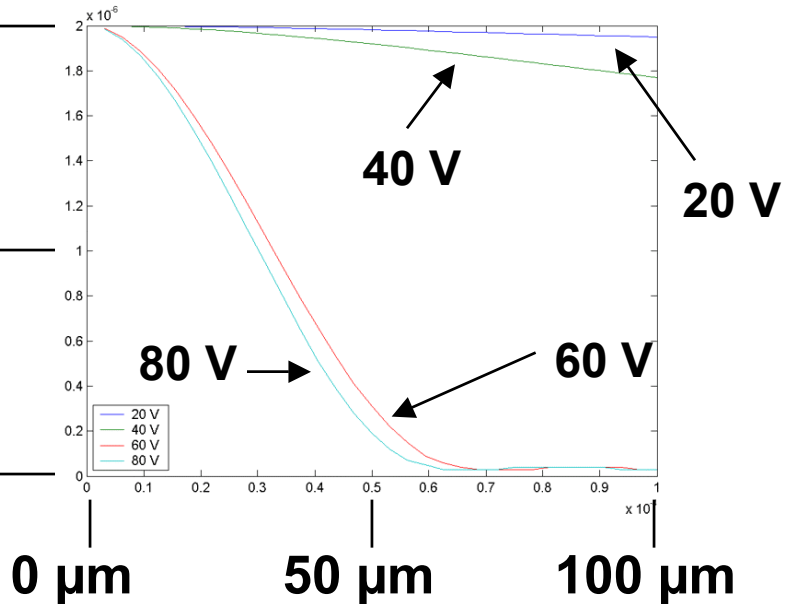
32 NODAS linear beam and gap elements



2 μm

1 μm

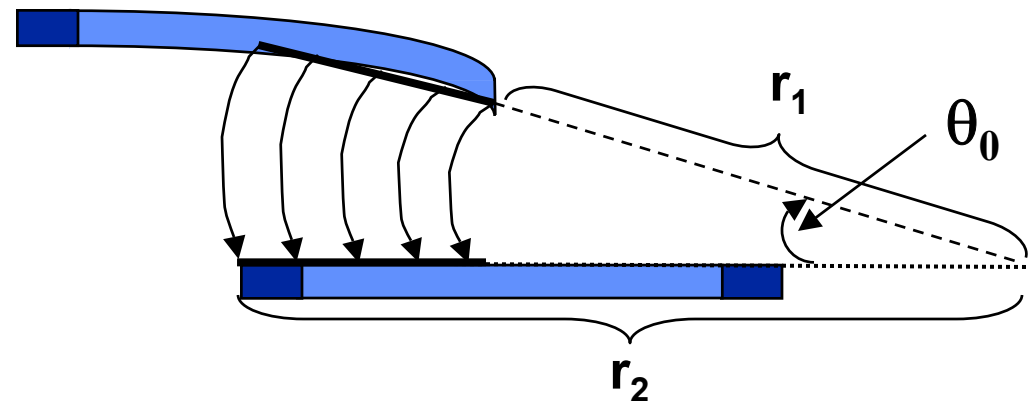
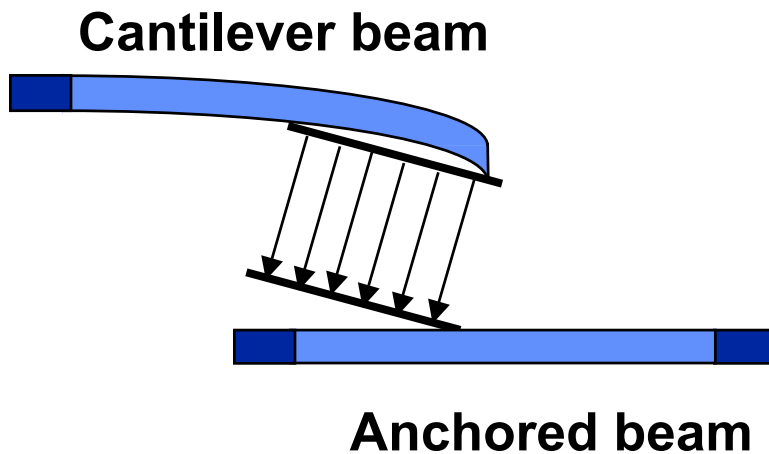
0 μm



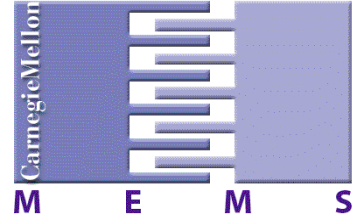
Beam Mode Shapes

Future Work

- Rotated parallel plate approximation
 - Valid when angles of beam electrodes match closely
 - Inaccurate when angle between beam electrodes different
- Use radial electric field between electrodes
- A-priori analytically integrate Taylor Series model of F_e



Acknowledgements



- DARPA and US Army CECOM
- Pittsburgh Digital Greenhouse